- Fold-and-thrust belts and submarine accretionary wedges share these features:
 - In cross-section, they occupy a wedge-shaped deformed region overlying a basal detachment or décollement fault.
 - The rocks or sediments beneath this fault show very little deformation.
 - The décollement fault characteristically dips toward the interior of the mountain belt or, in the case of a submarine wedge, roward the island arc
 - The topography, in contrast, slopes toward the toe or deformation front of the wedge.
 - Deformation within the wedge is generally dominated by imbricate thrust faults verging toward the toe and related fault-bend folding.

Examples of fold-and-thrust belts and accretionary wedges: Canadian Rocky Mountains and the Lesser Antilles





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(Buiter, Tectonophysics, 2012)

The starting point of the mechanical theory for these structures is the recognition that they are analogous to a wedge of sand in front of a moving bulldozer.



The sand, rock, or sediment deforms until it develops a constant critical taper: i.e., the wedge slides stably without further deformation as it is pushed unless it encounters new fresh materials at the toe.

- The geometry of the critical taper is governed by the relative magnitude of the basal frictional resistance to internal strength.
- An increase in the basal resistance increases the critical taper while an increase in the wedge strength decreases the critical taper.
- The state of stress within a critically tapered wedge in the upper crust is everywhere on the verge of Coulomb failure since the taper is a product of continued brittle deformation.

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Mechanics of a Bulldozer Wedge: Kinematics



- $\alpha + \beta$: the critical taper.
 - α : the surface slope of a deformed wedge
 - β : the slope of a rigid hillside.
- From the mass conservation, we get the rate of the wedge growth for a constant density ρ:

$$\frac{d}{dt}\left[\frac{1}{2}\rho W^2 \tan(\alpha+\beta)\right] = \rho hv. \tag{1}$$

Mechanics of a Bulldozer Wedge: Kinematics

By the definition of the critical taper, $\alpha + \beta$ does not change in time. So, (1) becomes

$$W\frac{dW}{dt} = \frac{hv}{\tan(\alpha + \beta)}.$$
 (2)

The solution is

$$W = \left[\frac{2hv}{\tan(\alpha+\beta)}\right]^{1/2} t^{1/2} \approx \left[\frac{2hv}{\alpha+\beta}\right]^{1/2} t^{1/2}, \tag{3}$$

where the approximation is valed if $\alpha + \beta \ll 1$ in radian.

- Since the coefficient is constant according to the assumptions we made, both the width and height(= $W \tan(\alpha + \beta)$) grow like $t^{1/2}$.
- The growth is self-similar in the sense that the wedge at time 2*t* is indistinguishable from the wedge at time t, magnified $\sqrt{2}$ times.



- In the setting described above, we consider the force balance on an infinitesimal segment of the wedge lying between x and x + dx.
- First, a gravitational body force whose x component per unit length along strike is

$$F_g = -(
ho H dx)g\sinlpha,$$
 (4)

where g is the acceleration of gravity, and H is the local wedge thickness.

Second, there is the net force exerted by the compressive tractions σ_{xx} acting on the sidewalls at x and x + dx. Setting compressive stress to be negative, we get this force as

$$F_{s} = \int_{0}^{H} t_{x}(x) + t_{x}(x + dx)dz$$

=
$$\int_{0}^{H} (\sigma(x) \cdot -\mathbf{e}_{x})_{x} + (\sigma(x + dx) \cdot \mathbf{e}_{x})_{x}dz$$

=
$$\int_{0}^{H} [-\sigma_{xx}(x, z) + \sigma_{xx}(x + dx, z)]dz,$$
 (5)

where $\mathbf{e}_x = (1, 0)$ is the unit basis vector for the *x* axis.

- ► Thirdly, and finally, there is the surface force exerted on the base.
- ► In a coordinate system of which x' axis is parallel to the bottom surface, we get the traction vector, $\mathbf{t}' = \boldsymbol{\sigma}' \cdot (0, 1) = (\sigma_{x'z'}, \sigma_{z'z'}) \equiv (\tau_b, \sigma_n)$.
- Trasnforming t' to the x z coordinate system,

$$\begin{bmatrix} t_x \\ t_z \end{bmatrix} = \begin{pmatrix} \cos(\alpha + \beta) & -\sin(\alpha + \beta) \\ \sin(\alpha + \beta) & \cos(\alpha + \beta) \end{pmatrix} \begin{bmatrix} \tau_b \\ \sigma_n \end{bmatrix},$$
 (6)

we get

$$F_b = t_x dx = [\tau_b \cos(\alpha + \beta) - \sigma_n \sin(\alpha + \beta)] dx.$$
(7)

► The base is governed by a frictional sliding condition, $\tau_b = \mu_b(-\sigma_n)$, where μ_b is the basal friction coefficient. The sign is to make traction acting on +x direction when σ_n is compressive and thus negative. With this, we have

$$F_{b} = -\sigma_{n} \left[\mu_{b} \cos(\alpha + \beta) + \sin(\alpha + \beta) \right] dx.$$
(8)

The force balance conditions is

$$F_g + F_s + F_b = 0. \tag{9}$$

- The first two forces, F_g and F_s , act in the -x direction, whereas F_b acts in the +x direction.
- We divide (9) with dx and assume $dx \rightarrow 0$. The result is

$$-\rho gH\sin\alpha - \sigma_n \left[\mu_b \cos(\alpha + \beta) + \sin(\alpha + \beta)\right] + \frac{d}{dx} \int_0^H \sigma_{xx} dz = 0.$$
(10)

For α ≪ 1 and β ≪ 1, we employ the approximations sin α ≈ α, sin(α + β) ≈ α + β, cos(α + β) ≈ 1 and σ_n ≈ −ρgH. This reduces (10) to

$$\rho g H(\beta + \mu_b) + \frac{d}{dx} \int_0^H \sigma_{xx} dz \approx 0.$$
 (11)

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> The failure criterion for non-cohesive dry sand can be written in the form

$$\frac{\sigma_1}{\sigma_3} = \frac{1 + \sin\phi}{1 - \sin\phi},\tag{12}$$

where σ_1 and σ_3 are the greatest and least principal compressive stresses, respectively, and ϕ is the internal friction angle.

From the last class, we know that frictional sliding satisfies $\sigma_s = \tan \phi \sigma_n$ (cohesion is zero since non-cohesive!) and also

$$\sigma_s = \frac{\sigma_1 - \sigma_3}{2} \sin 2\theta = \frac{\sigma_1 - \sigma_3}{2} \cos \phi, \tag{13}$$

$$\sigma_n = \frac{\sigma_1 + \sigma_3}{2} - \frac{\sigma_1 - \sigma_3}{2}\sin 2\theta = \frac{\sigma_1 + \sigma_3}{2} - \frac{\sigma_1 - \sigma_3}{2}\cos\phi, \quad (14)$$

since $2\theta = \pi/2 - \phi$ under the failure condition.

Plugging (13) and (14) into the sliding condition, we get (12).

▶ In a narrow taper (i.e., $\alpha \ll 1$ and $\beta \ll 1$), σ_1 and σ_3 are approximately horizontal and vertical:

$$\sigma_{xx} \approx \sigma_1 \approx -\frac{1 + \sin \phi}{1 - \sin \phi} \rho gz, \qquad (15)$$

$$\sigma_{zz} \approx \sigma_3 \approx -\rho gz. \qquad (16)$$

The traction acting on the sidewalls then reduces in this approximation to

$$\frac{d}{dx}\int_{0}^{H}\sigma_{xx}dz\approx-\frac{1+\sin\phi}{1-\sin\phi}\rho gH(\alpha+\beta), \tag{17}$$

where we have used the relation $dH/dx = tan(\alpha + \beta) \approx \alpha + \beta$.

Substituting (17) into (11), we obtain the approximate critical taper equation for a dry sand wedge in front of a bulldozer:

$$\alpha + \beta \approx \frac{1 + \sin \phi}{1 - \sin \phi} (\beta + \mu_{\mathcal{B}}).$$
(18)

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• The critical taper, $\alpha + \beta$ is proportional to the basal friction coefficient, μ_b ; inversely to the internal friction angle, ϕ .

- Now, we want to consider the effects of pore fluid pressure on the stability of the wedge.
- Also, we want to get exact solutions.
- Here is the problem setting:



The static equilibrium equation is

$$\frac{\partial \sigma_{xx}}{\partial x} + \frac{\partial \sigma_{xz}}{\partial z} - \rho gz \sin \alpha = 0,$$
(19)
$$\frac{\partial \sigma_{xz}}{\partial x} + \frac{\partial \sigma_{zz}}{\partial z} + \rho gz \cos \alpha = 0.$$
(20)

Boundary conditions on the upper surface of the wedge:

$$\sigma_{xz} = \mathbf{0}, \quad \sigma_{zz} = -\rho_f g \mathcal{D}, \tag{21}$$

where *D* is the water depth.

It is convenient to define the generalized Hubbert-Rubey pore fluid to lithostatic pressure ratio by

$$\lambda = \frac{p_f - \rho_f g D}{-\sigma_{zz} - \rho_f g D}.$$
(22)

• We assume that λ , ρ and the internal friction coefficient (μ) are all constant.

- The density ρ is actually defined as ρ = (1 − η)ρ_s + ηρ_f. Since the rock (ρ_s) and fluid density (ρ_f) are constant, the constant aggregate density (i.e. constant ρ) implies a constant porosity, η.
- For convenience, we need to work out some more relations between stress components and other parameters.



From geometry,

$$\rho^* = -\frac{1}{2}(\sigma_{zz}^* + \sigma_{xx}^*), \tag{23}$$

$$\boldsymbol{R} = \boldsymbol{p}^* \sin \phi + \boldsymbol{S} \cos \phi, \qquad (24)$$

$$\sigma_{xx}^* = -\boldsymbol{p}^* - \boldsymbol{R}\cos 2\psi, \qquad (25)$$

$$\sigma_{zz}^* = -\boldsymbol{p}^* + \boldsymbol{R}\cos 2\psi, \qquad (26)$$

$$\sigma_{xz}^* = R\sin 2\psi. \tag{27}$$

From these relations, we derive the following expressions for later uses:

$$\frac{\sigma_{zz}^{*} - \sigma_{xx}^{*}}{2} = -\frac{\sigma_{zz}^{*} + \sigma_{xx}^{*}}{2} \sin \phi \cos 2\psi + S \cos \phi \cos 2\psi$$
$$\Rightarrow \frac{\sigma_{zz}^{*} - \sigma_{xx}^{*}}{2} = \frac{S \cot \phi - \sigma_{zz}^{*}}{\csc \phi \sec 2\psi - 1}.$$
(28)

$$\sigma_{xz}^* = \tan 2\psi \frac{S \cot \phi - \sigma_{zz}^*}{\csc \phi \sec 2\psi - 1}.$$
(29)

The following stress components satisfy the static momentum balance, the failure condition and the boundary conditions on the top surface:

$$\sigma_{xz}^* = (\rho - \rho_f) gz \sin \alpha, \tag{30}$$

$$\sigma_{zz}^* = -\rho_f g D - \rho g z \cos \alpha, \tag{31}$$

$$\sigma_{xx}^* = -\rho_f g D - \rho g z \cos \alpha \frac{\csc \phi \sec 2\psi_0 - 2\lambda + 1}{\csc \phi \sec 2\psi_0 - 1},$$
(32)

provided that

$$\frac{\tan 2\psi_0}{\csc\phi\sec 2\psi_0 - 1} = \left(\frac{1 - \rho_f/\rho}{1 - \lambda}\right)\tan\alpha.$$
(33)

(33) relates the stress orientation angle ψ₀ to the surface slope α; we have assumed that ψ₀ is constant and have made use of the relation dD/dx = -sin α.

- (30), (31), (32) are an exact solution for the state of stress in a sloping half-space on the verge of Coulomb failure.
- All that remains is to satisfy the basal boundary condition. Allowing for a different pore-fluid regime (like a different porosity) on the décollement fault, we have this boundary condition for the base:

$$\tau_b = -\mu_b(\sigma_n + \rho_{fb}), \tag{34}$$

where p_{fb} is the pore-fluid pressure on the base and μ_b is the basal coefficient of friction.

Expressing the basal shear and normal stress in terms of stress components in the x – z coordinate system:

$$\tau_b = 1/2(\sigma_{zz}^* - \sigma_{xx}^*)\sin 2(\alpha + \beta) + \sigma_{xz}^*\cos 2(\alpha + \beta), \tag{35}$$

$$\sigma_n = \sigma_{zz}^* - \sigma_{xz}^* \sin 2(\alpha + \beta) - 1/2(\sigma_{zz}^* - \sigma_{xx}^*)[1 - \cos 2(\alpha + \beta)].$$
(36)

- (30), (31), (32), (35) and (36) are used to determine the dip of the surface on which the frictional sliding condition (34) is satisfied.
- After some algebra, β is given by

$$\alpha + \beta = \psi_b - \psi_0, \tag{37}$$

where

$$\frac{\tan 2\psi_b}{\csc \phi \sec 2\psi_b - 1} = \mu_b \left(\frac{1 - \lambda_b}{1 - \lambda}\right). \tag{38}$$

 (37) is the exact critical taper equation for a homogeneous noncohesive Coulomb wedge.

- Here is the usual procedure we take to play with the above critical taper equation.
 - 1. We can start from (38) to find ψ_b from μ_b and λ_b .

$$rac{ ext{tan 2}\psi_b}{\csc \phi \sec 2\psi_b - 1} = \mu_b \left(rac{ ext{1} - \lambda_b}{ ext{1} - \lambda}
ight).$$

2. Then, assume a value of the surface slope, α , to get ψ_0 from (33).

$$\frac{\tan 2\psi_0}{\csc \phi \sec 2\psi_0 - 1} = \left(\frac{1 - \rho_f / \rho}{1 - \lambda}\right) \tan \alpha.$$

3. Since ψ_b , ψ_0 and α are known, we get β from the taper equation, (37).

$$\beta = \psi_{\mathbf{b}} - \psi_{\mathbf{0}} - \alpha.$$

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Once we get all these angles, we can predict the orientation of step-up from the basal décollement fault as well as that of backthrust faults.



Noncohesive Could



Fig. 9. Diagram showing stable and unstable regions of dry sand wedges having $\varphi = 30^{\circ}$ and $\varphi_b = 10^{\circ}$. Critical wedges labeled 1-8 are depicted in Figure 10.



6. combined normal and

7. surface at angle